

A Hosford-based orthotropic plasticity model in LS-DYNA

Filipe Andrade¹, Thomas Borrrell², Paul DuBois³, Markus Feucht⁴

¹DYNAmore GmbH

²DYNAmore Nordic AB

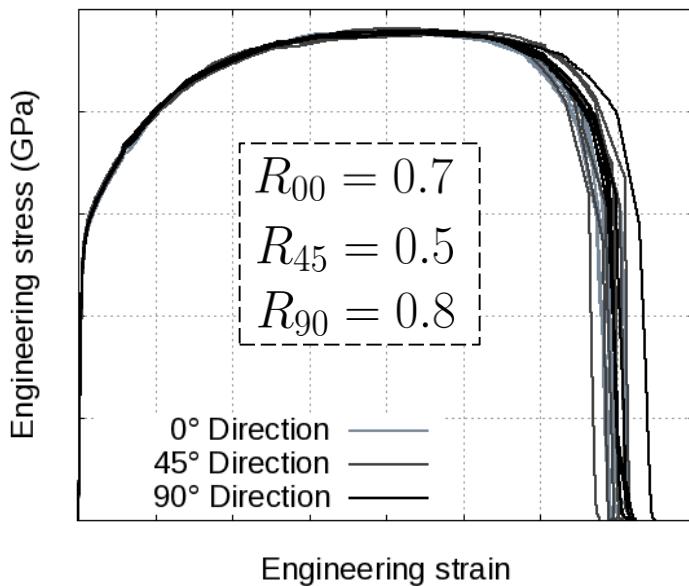
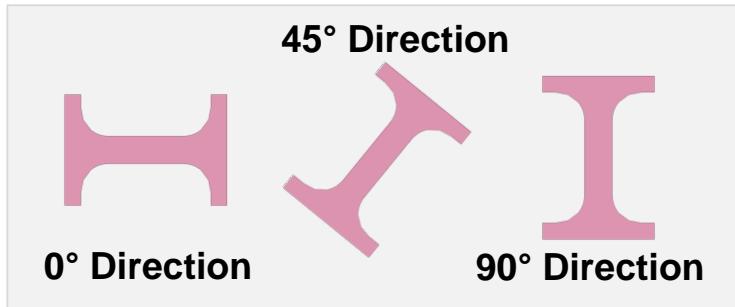
³Consultant

⁴Daimler AG

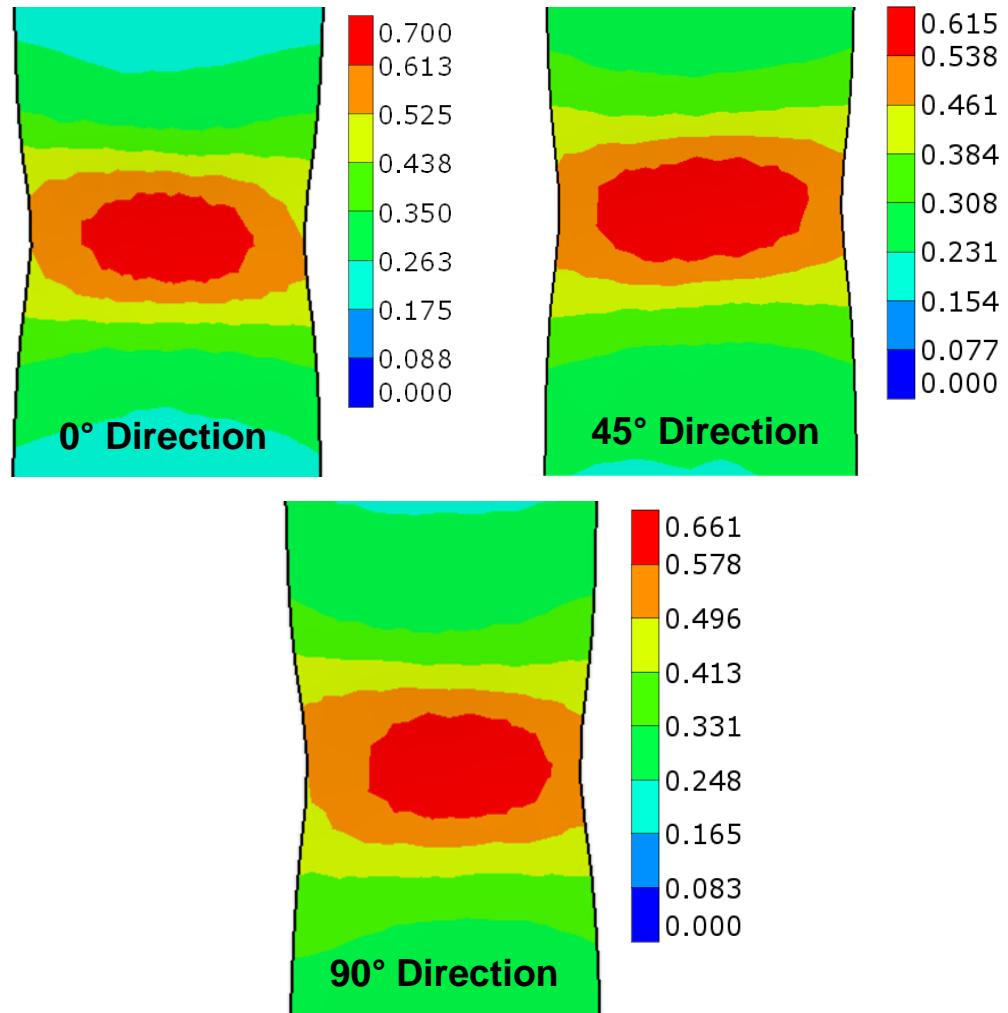
October 17th 2018

Orthotropic behavior

Example: aluminum sheet



Strain fields prior to failure (DIC measurement)

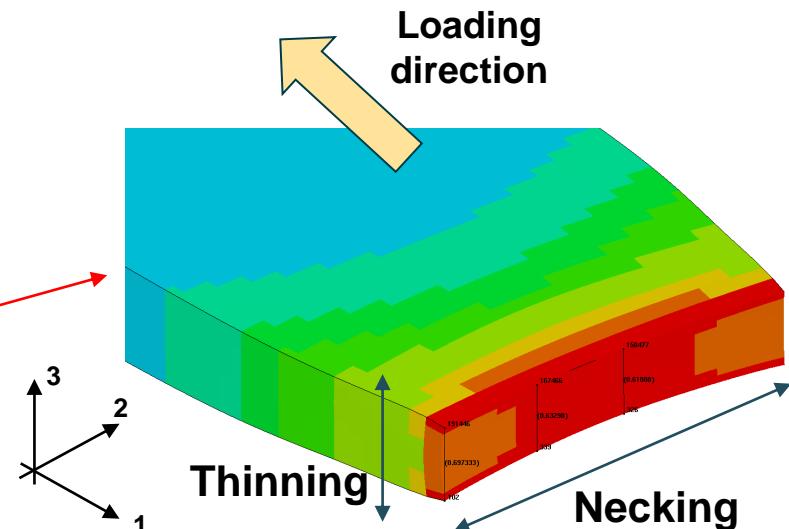
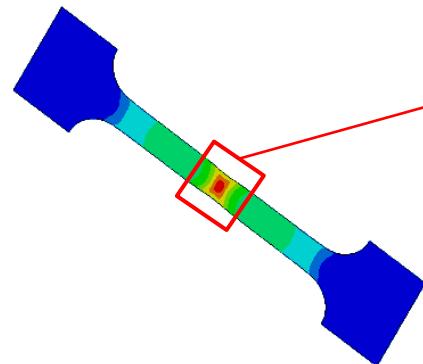


Orthotropic behavior

The Lankford parameter (R value)

- Definition (uniaxial tension):

$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p}$$



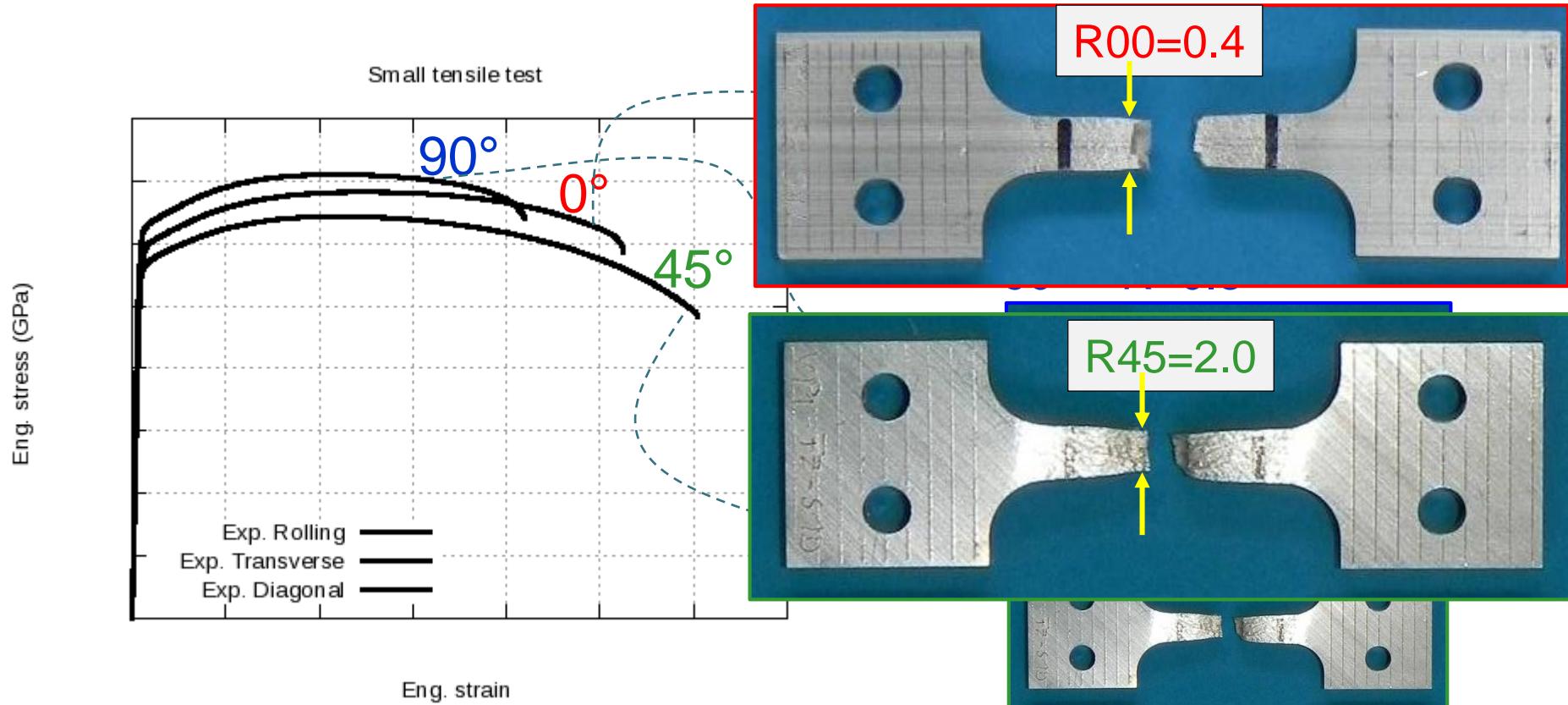
$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p} = 1.0 \longrightarrow \dot{\varepsilon}_{22}^p = \dot{\varepsilon}_{33}^p \longrightarrow \text{Thinning and necking are comparable}$$

$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p} < 1.0 \longrightarrow \dot{\varepsilon}_{22}^p < \dot{\varepsilon}_{33}^p \longrightarrow \text{More thinning} \quad \text{Less necking}$$

$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p} > 1.0 \longrightarrow \dot{\varepsilon}_{22}^p > \dot{\varepsilon}_{33}^p \longrightarrow \text{Less thinning} \quad \text{More necking}$$

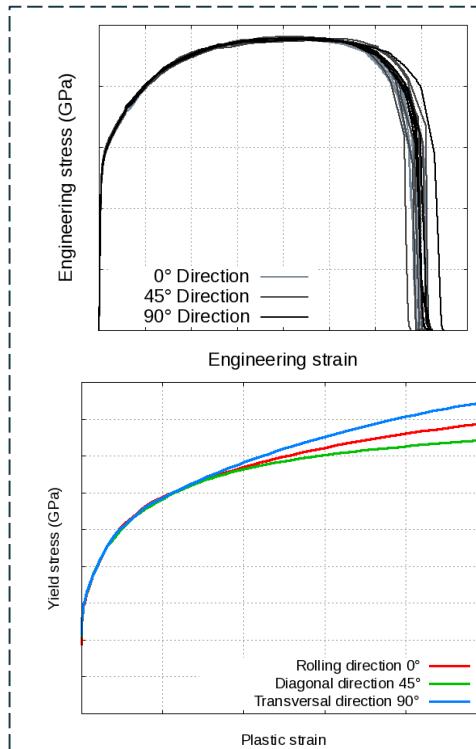
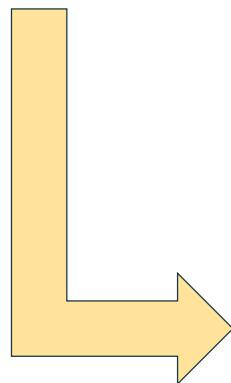
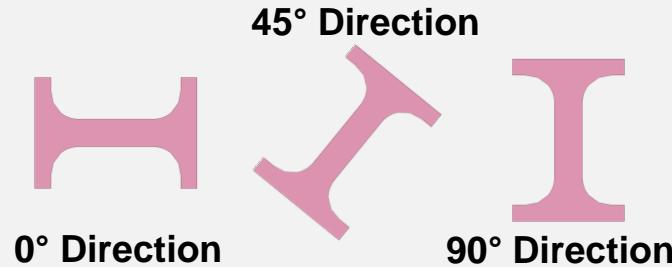
Orthotropic behavior

Example: aluminum extrusion

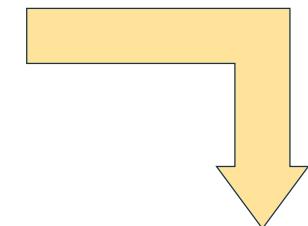


Orthotropic modeling

From experimental data to simulation parameters



$$\begin{aligned}R_{00} &= 0.7 \\R_{45} &= 0.5 \\R_{90} &= 0.8 \\R_B &= \dots \\R_S &= \dots\end{aligned}$$



LS-DYNA



MATERIAL MODELING IN LS-DYNA

*MAT_036 + HR=3

Barlat & Lian (1989)

R values are similar

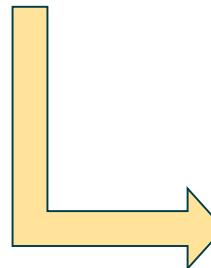
*MAT_3-PARAMETER_BARLT						HR	P1	P2	ITER
\$	MID	RO	E	PR		3	E0	SPI	P3
\$	M	R00	R45	R90					
\$	1	2.70E-06	70.0	0.3					
\$	8.0	0.8	1.0	0.9					
\$...									

$$\Phi(\boldsymbol{\sigma}) = \frac{1}{2} (a |K_1 + K_2|^m + a |K_1 - K_2|^m + c |2K_2|^m) - \sigma_y^m = 0$$

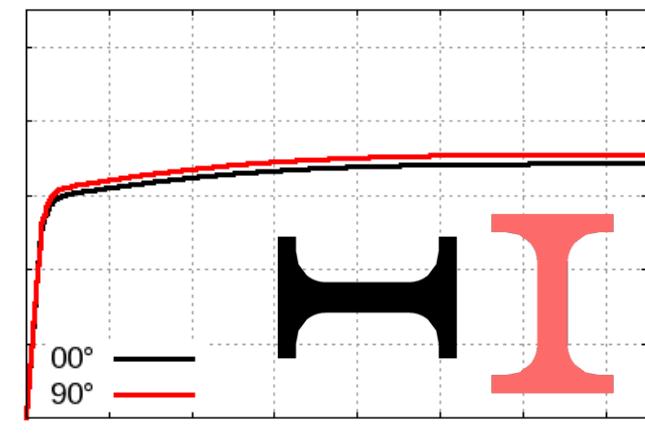
$$\begin{aligned} R_{00} &= 0.8 \\ R_{45} &= 1.0 \\ R_{90} &= 0.9 \\ \sigma_y, m & \end{aligned}$$

internal fitting

$$\begin{aligned} a &= \dots \\ c &= \dots \\ h &= \dots \\ p &= \dots \end{aligned}$$



Eng. stress



Eng. strain

*MAT_036 + HR=3

Barlat & Lian (1989)

R values differ significantly from each other

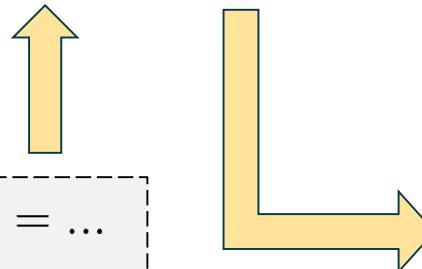
*MAT_3-PARAMETER_BARLT						HR	P1	P2	ITER
\$	MID	RO	E	PR					
\$	1	2.70E-06	70.0	0.3					
\$	M		R00	R45	R90				
\$	8.0		0.5	1.0	2.0	LCID			
\$...						3	E0	SPI	P3

$$\Phi(\boldsymbol{\sigma}) = \frac{1}{2} (a |K_1 + K_2|^m + a |K_1 - K_2|^m + c |2K_2|^m) - \sigma_y^m = 0$$

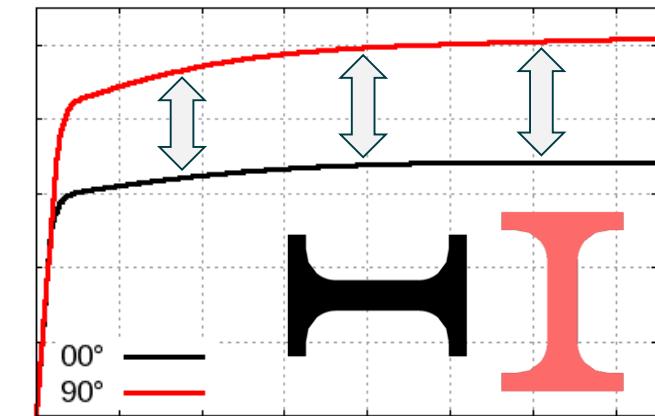
$$\begin{aligned} R_{00} &= 0.5 \\ R_{45} &= 1.0 \\ R_{90} &= 2.0 \\ \sigma_y, m & \end{aligned}$$

internal fitting

$$\begin{aligned} a &= \dots \\ c &= \dots \\ h &= \dots \\ p &= \dots \end{aligned}$$



Eng. stress



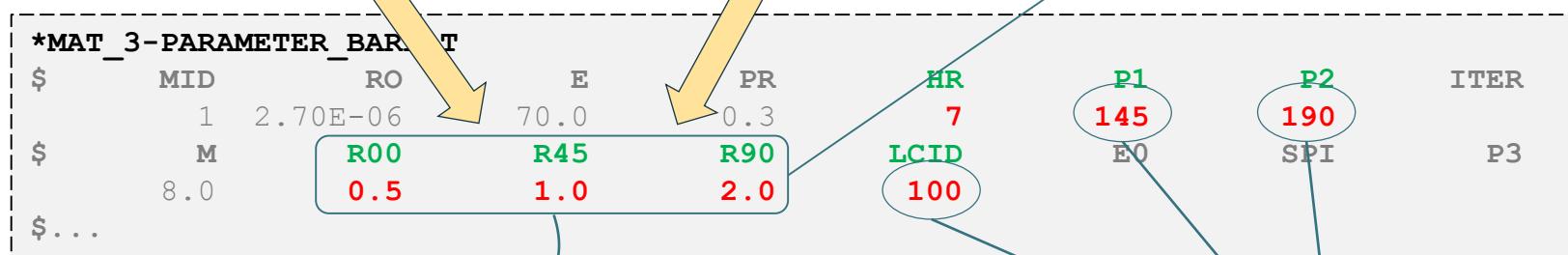
Eng. strain

*MAT_036 + HR=7

Fleischer & Borrvall (2007): modified Barlat & Lian

R values differ significantly from each other

$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p}$$

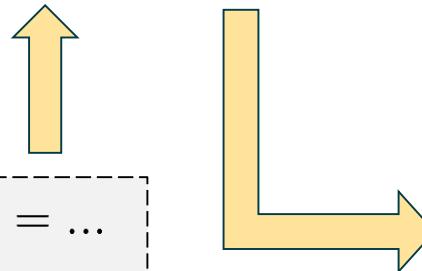


$$\Phi(\boldsymbol{\sigma}) = \frac{1}{2} (a |K_1 + K_2|^m + a |K_1 - K_2|^m + c |2K_2|^m) - \sigma_y^m = 0$$

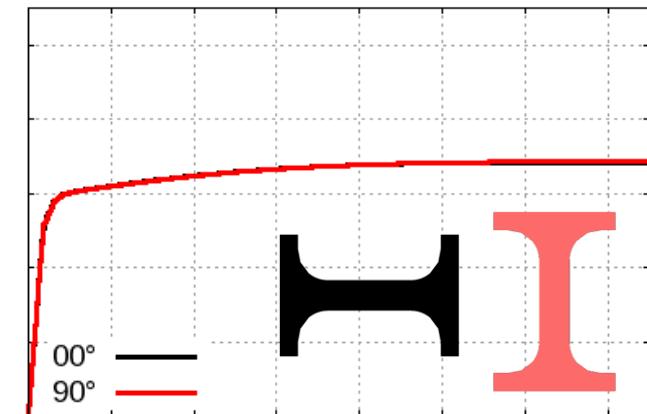
$$\begin{aligned} R_{00} &= 0.5 \\ R_{45} &= 1.0 \\ R_{90} &= 2.0 \\ \sigma_y, m & \end{aligned}$$

internal fitting

$$\begin{aligned} a &= \dots \\ c &= \dots \\ h &= \dots \\ p &= \dots \end{aligned}$$



Eng. stress



Eng. strain

*MAT_036 + HR=7

Fleischer & Borrvall (2007): modified Barlat & Lian

Yield function

$$\Phi(\boldsymbol{\sigma}) = \frac{1}{2} (a |K_1 + K_2|^m + a |K_1 - K_2|^m + c |2K_2|^m) - \sigma_y^m = 0$$

$$\sigma_y(\boldsymbol{\sigma}, \varepsilon^p) = \alpha_{00}\sigma_y^{00}(\varepsilon^p) + \alpha_{45}\sigma_y^{45}(\varepsilon^p) + \alpha_{90}\sigma_y^{90}(\varepsilon^p) + \alpha_B\sigma_y^B(\varepsilon^p) + \alpha_{shear}\sigma_y^{shear}(\varepsilon^p)$$

Flow rule

$$\dot{\boldsymbol{\varepsilon}}^p = \begin{bmatrix} \dot{\varepsilon}_{11}^p & \dot{\varepsilon}_{12}^p & \dot{\varepsilon}_{13}^p \\ \dot{\varepsilon}_{21}^p & \dot{\varepsilon}_{22}^p & \dot{\varepsilon}_{23}^p \\ \dot{\varepsilon}_{31}^p & \dot{\varepsilon}_{32}^p & \dot{\varepsilon}_{33}^p \end{bmatrix} = \dot{\gamma} \left(\frac{\partial \Phi}{\partial \boldsymbol{\sigma}} + \Delta \mathbf{N} \right)$$

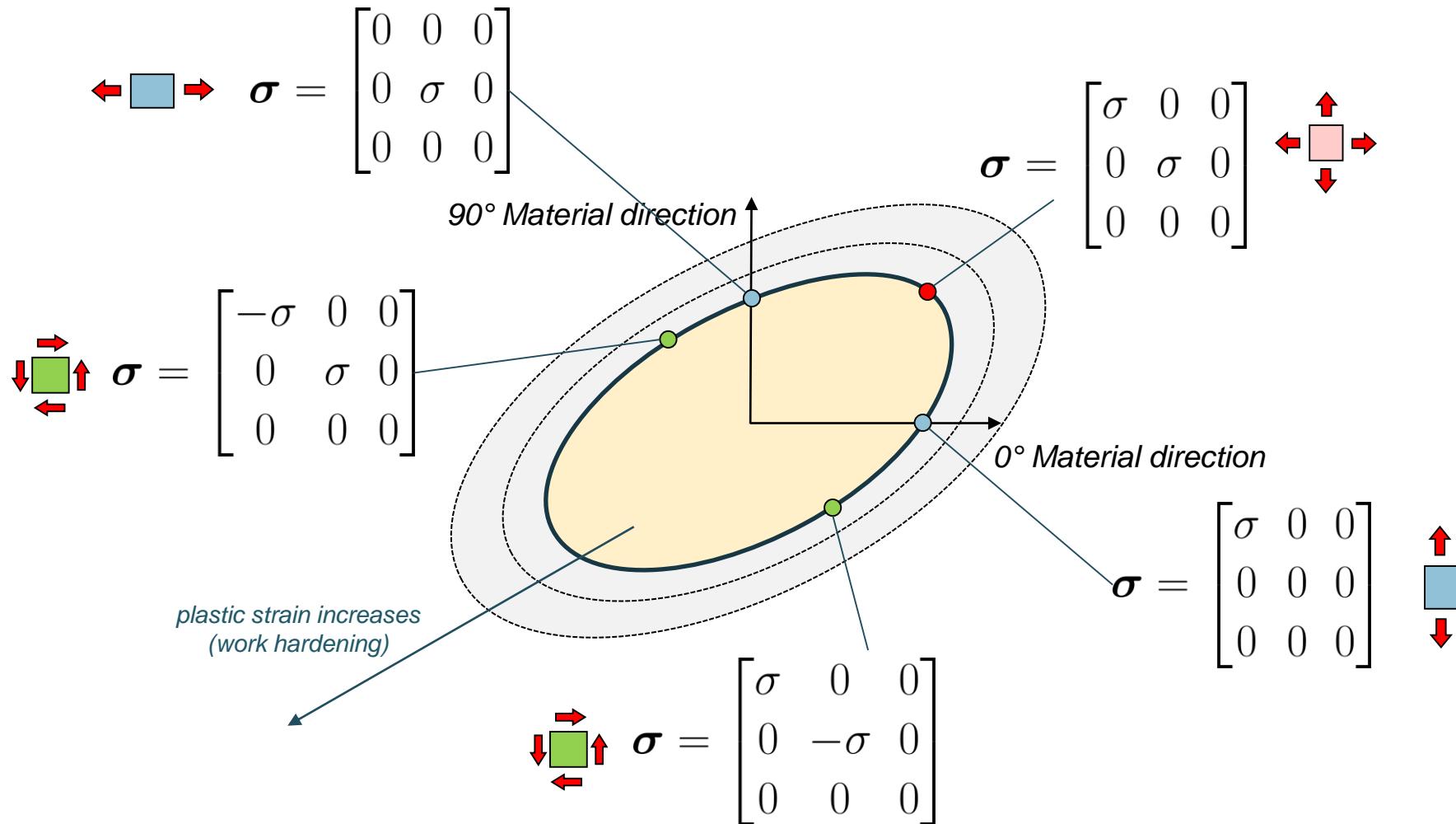
$$R = \frac{\dot{\varepsilon}_{22}^p}{\dot{\varepsilon}_{33}^p}$$

associated plasticity smallest necessary increment to match experimental R values

non-associated plasticity

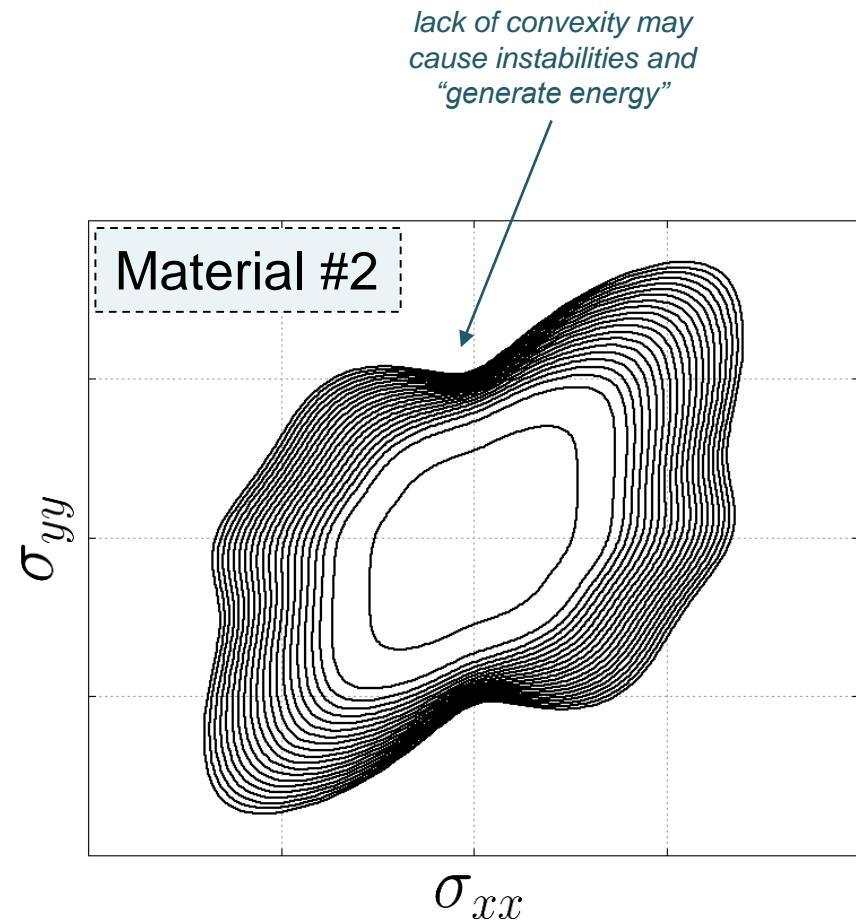
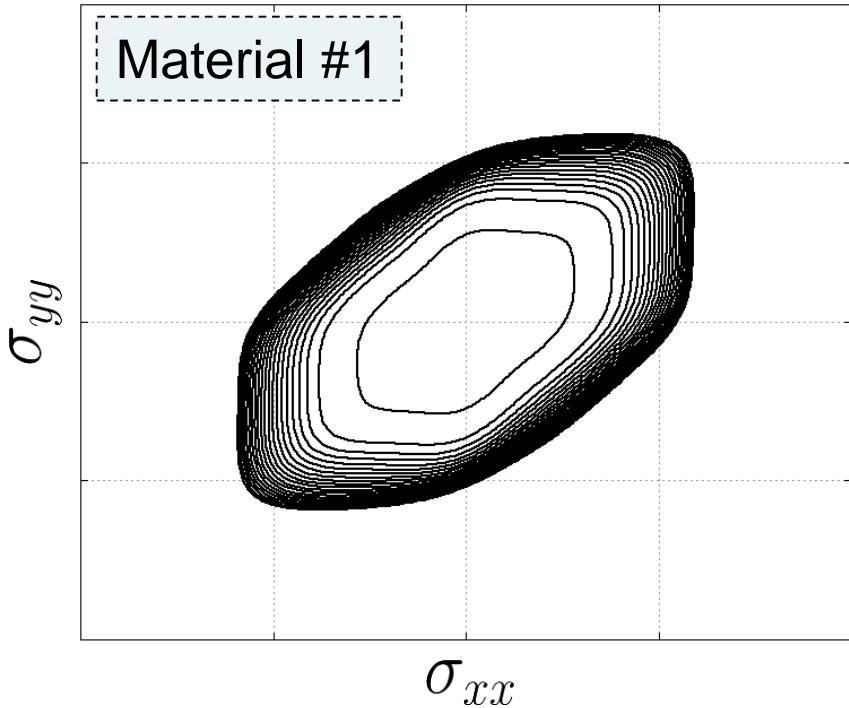
Yield surface (von Mises)

Example: Yielding at different stress states



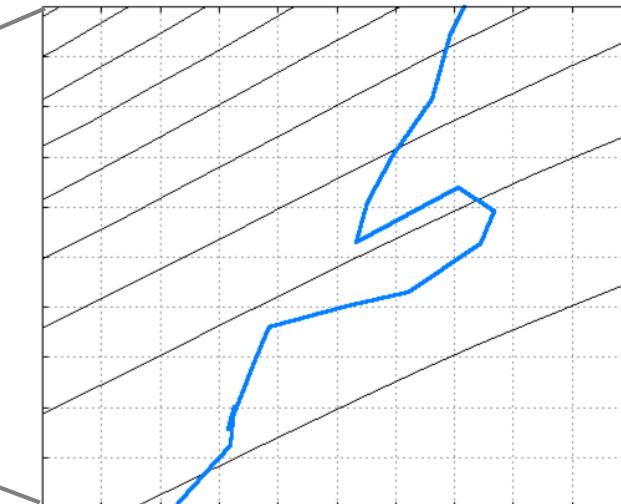
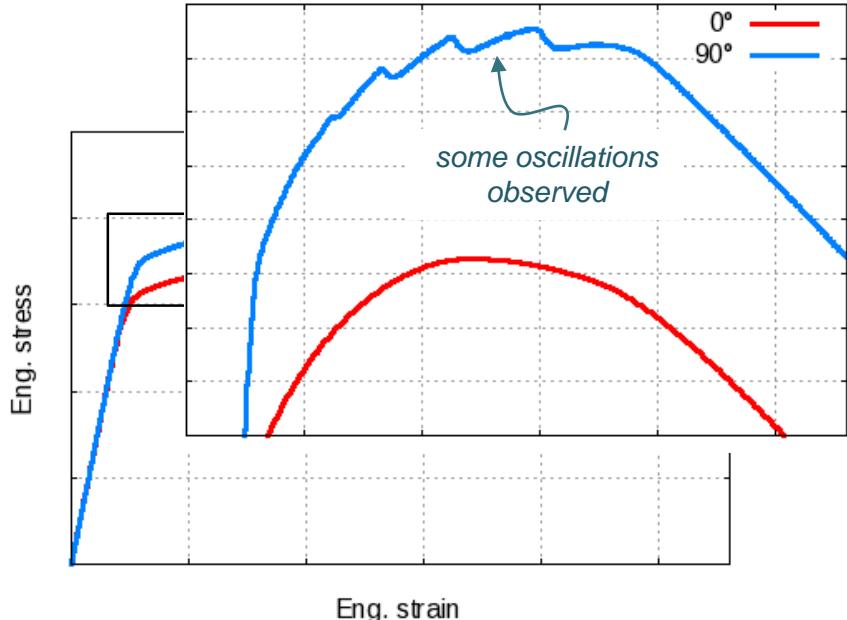
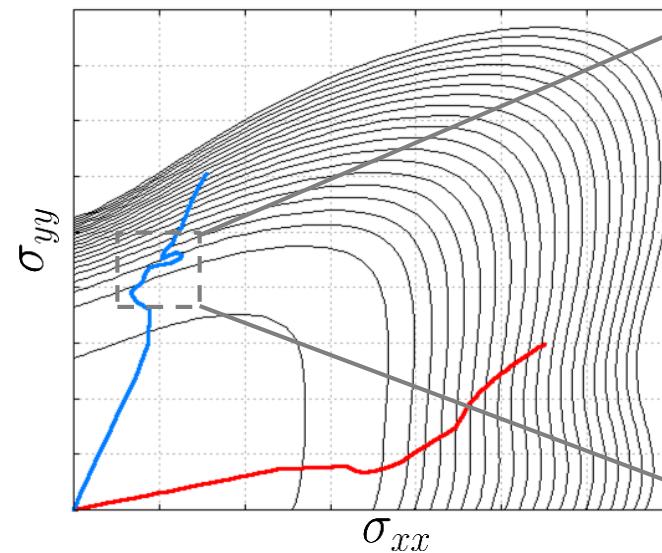
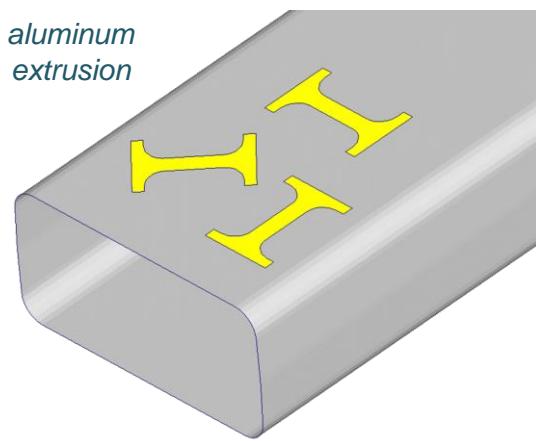
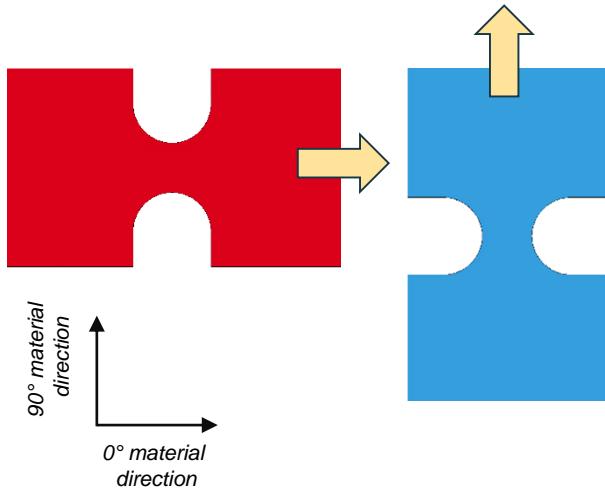
*MAT_036 + HR=7

Yield surface with increasing plastic strain



*MAT_036 + HR=7

Possible effect of a concave yield surface
Simulation of a notched tensile test



*MAT_036E + HOSF=0/1

Barlat- and Hosford-based orthotropic models

flow rule

$$\dot{\boldsymbol{\varepsilon}}^p = \dot{\gamma} \left(\frac{\partial \Phi}{\partial \boldsymbol{\sigma}} + \Delta \mathbf{N} \right)$$

*MAT_EXTENDED_3-PARAMETER_BARLAT / *MAT_036E					
\$	MID	RO	E	PR	
	1	2.70E-6	70.0	0.3	
\$	LCH00	LCH45	LCH90	LCHBI	LCHSH
	100	145	190		
\$	LCR00	LCR45	LCR90	LCRBI	LCRSH
	-0.5	-1.0	-2.0		
\$...					
					HOSF 0/1
					M 8

HOSF=0 $\rightarrow \Phi(\boldsymbol{\sigma}) = \frac{1}{2} (a |K_1 + K_2|^m + a |K_1 - K_2|^m + c |2K_2|^m) - \sigma_y^m = 0$
(*MAT_036+HR=7)

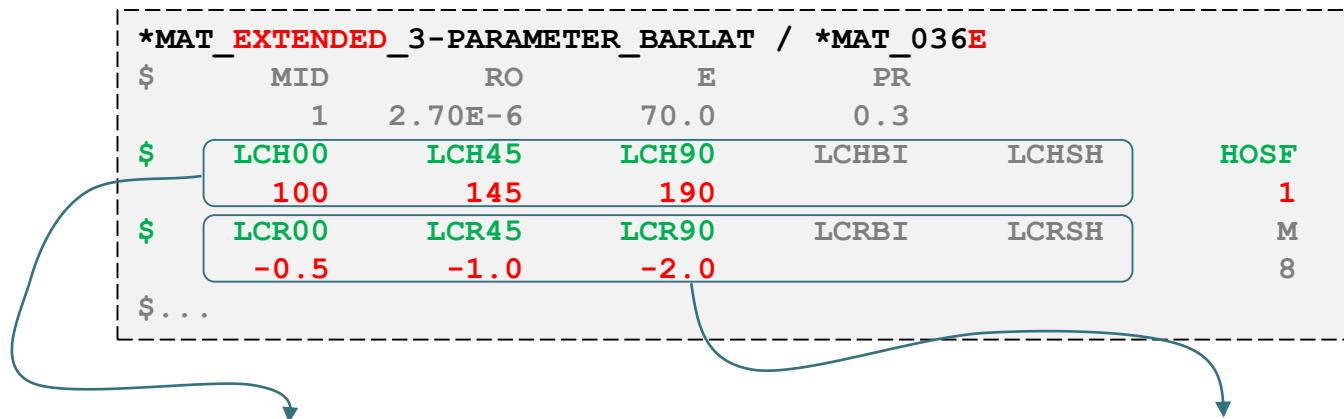
HOSF=1 $\rightarrow \Phi(\boldsymbol{\sigma}) = \frac{1}{2} (|\sigma_1|^m + |\sigma_2|^m + |\sigma_1 - \sigma_2|^m) - \sigma_y^m = 0$

No use of a , c , p and h in the yield function
(i.e., no influence of the R values on the yield function)

*MAT_036E + HOSF=1

Hosford-based orthotropic model

Yielding (hardening curve) and plastic flow
(R values) are treated separately!



Yield function

Flow rule

$$\Phi(\boldsymbol{\sigma}) = \sigma_{eff} - \sigma_y^m = 0$$

$$\sigma_{eff} = \frac{1}{2} (|\sigma_1|^m + |\sigma_2|^m + |\sigma_1 - \sigma_2|^m)$$

$$\dot{\boldsymbol{\varepsilon}}^p = \dot{\gamma} \left(\frac{\partial \Phi}{\partial \boldsymbol{\sigma}} + \Delta \mathbf{N} \right)$$

non-associated plasticity!

*MAT_036E + HOSF=1

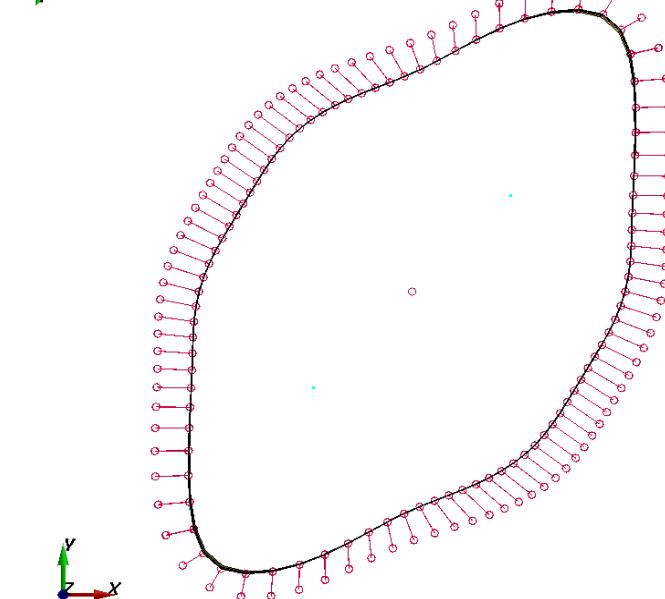
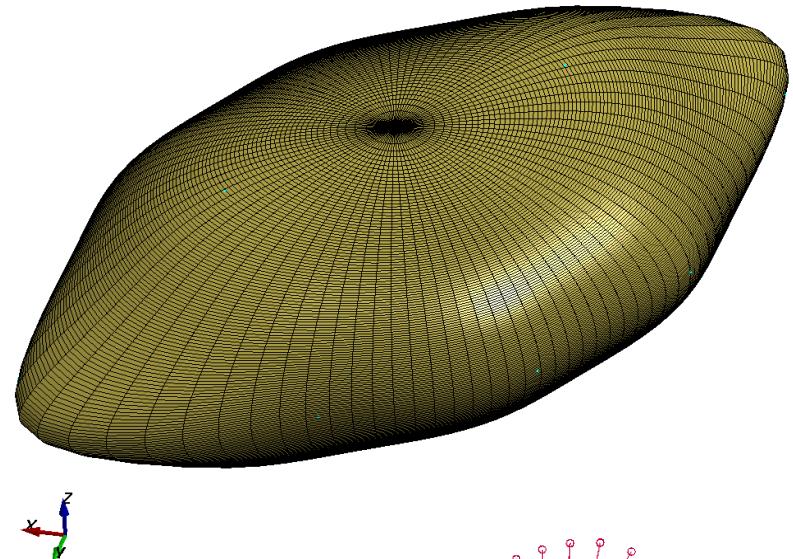
Hosford-based orthotropic model

$$\boldsymbol{\sigma} = \begin{bmatrix} \sigma_{11} & \sigma_{12} & 0 \\ 0 & \sigma_{22} & 0 \\ 0 & 0 & 0 \end{bmatrix}$$

Flow rule

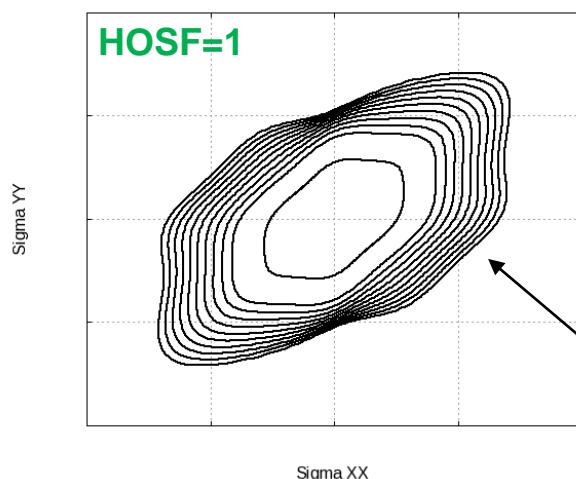
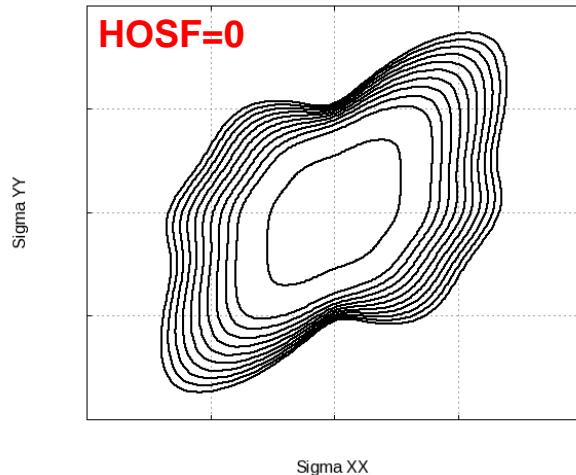
$$\dot{\boldsymbol{\varepsilon}}^p = \dot{\gamma} \left(\frac{\partial \Phi}{\partial \boldsymbol{\sigma}} + \Delta \mathbf{N} \right)$$

non-associated plasticity!

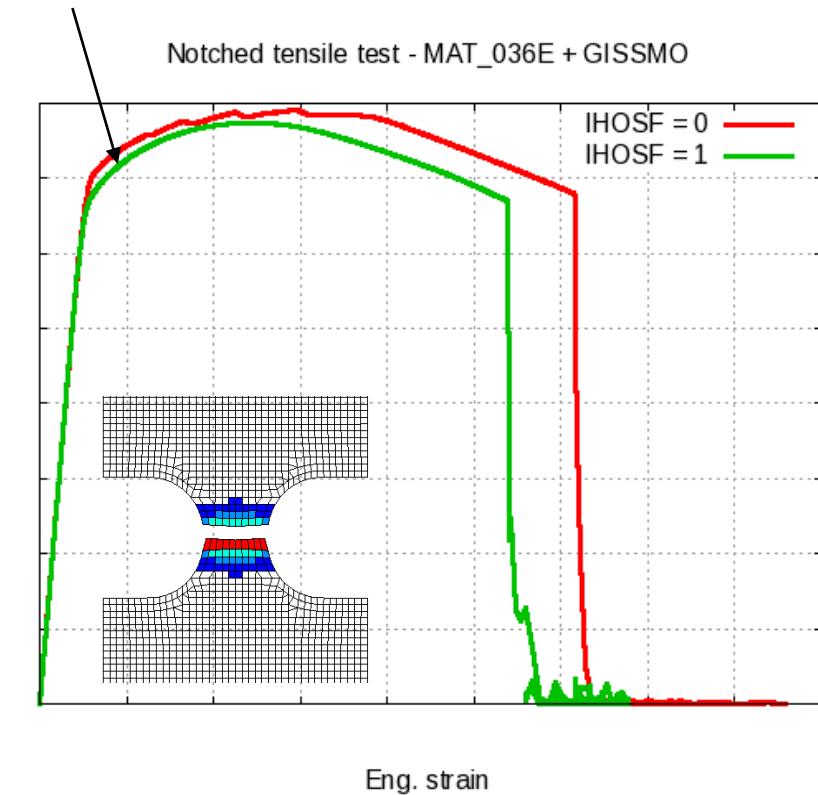


*MAT_036E + HOSF=0/1

Comparison between the formulations (aluminum extrusion)



No oscillations with HOSF=1



Yield surface is much more well-behaved with HOSF=1



EXAMPLE:

ALUMINUM SHEET ($t=1.5\text{mm}$)

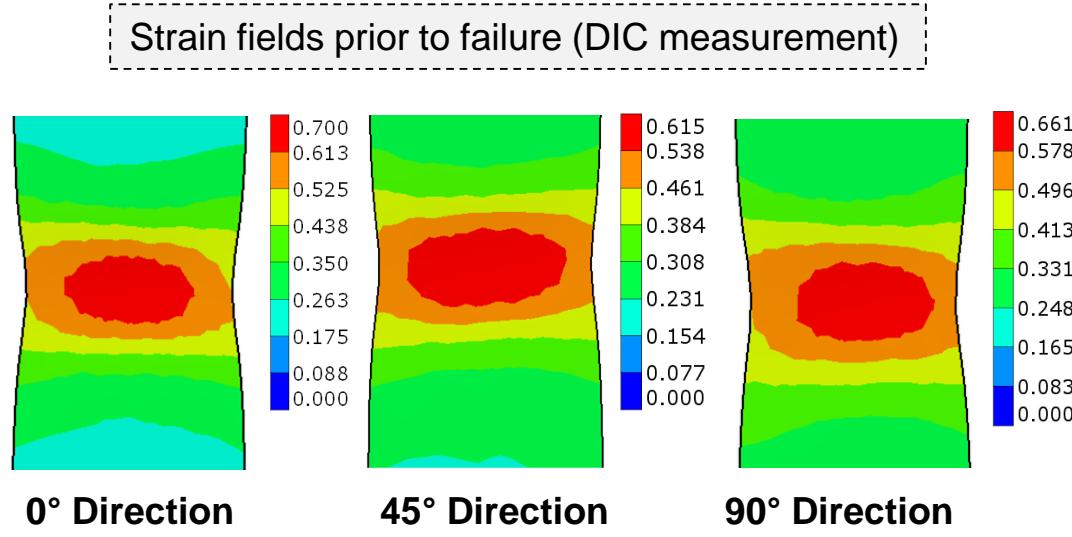
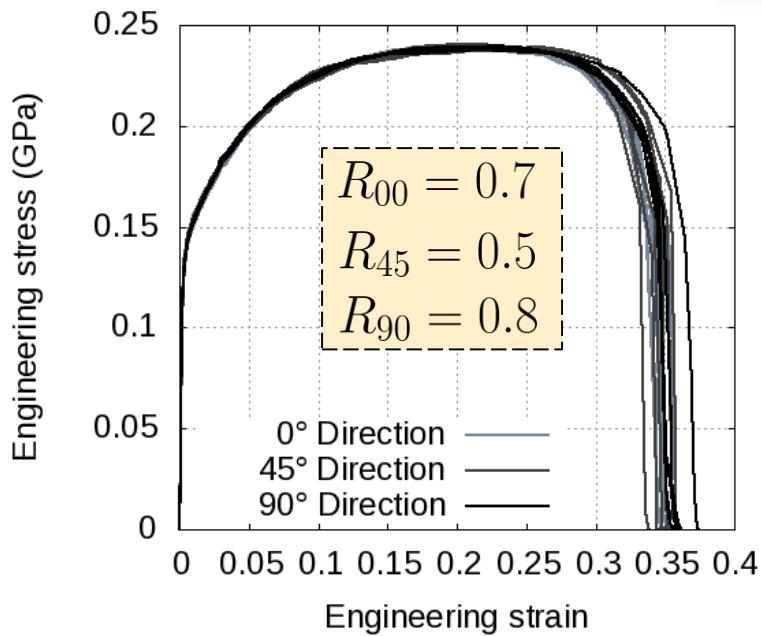
Aluminum sheet ($t=1.5\text{mm}$)

Experiments performed at DYNAmore in Stuttgart

Workshop:
“Material Characterization”
15th Oct 2018



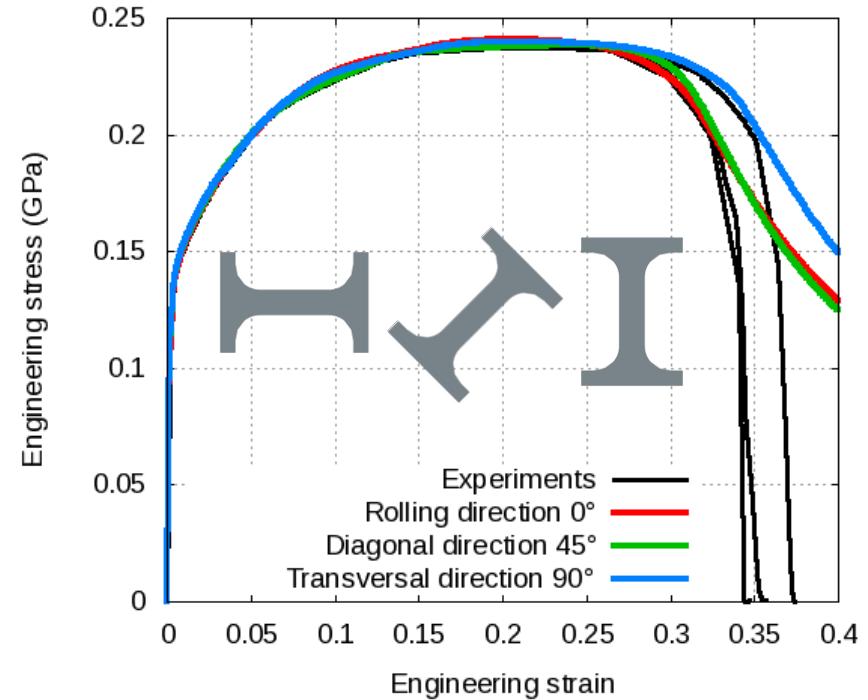
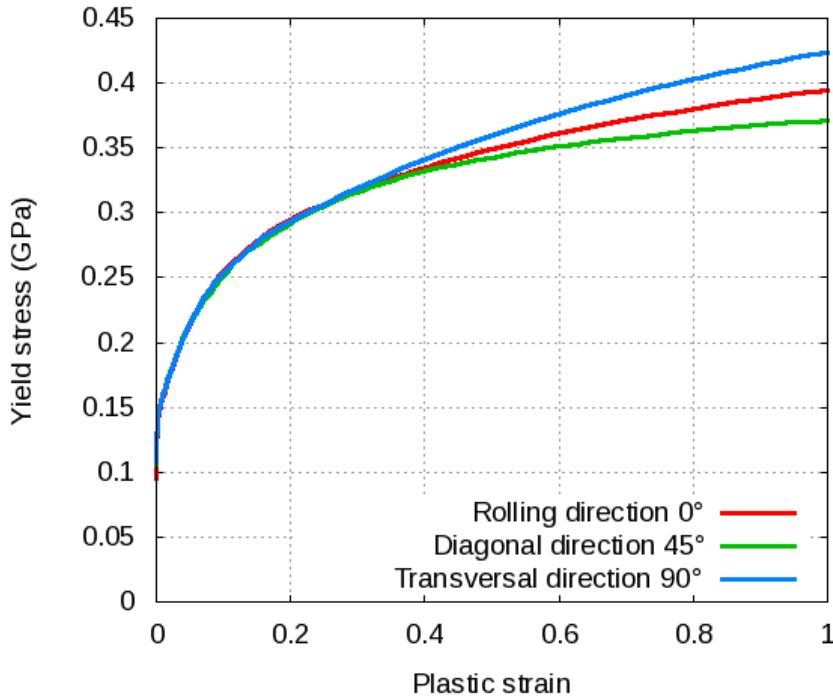
DYNAmore GmbH
Industriestr. 2
70565 Stuttgart



Hardening curves

Rolling (0°), diagonal (45°) and transversal (90°) directions

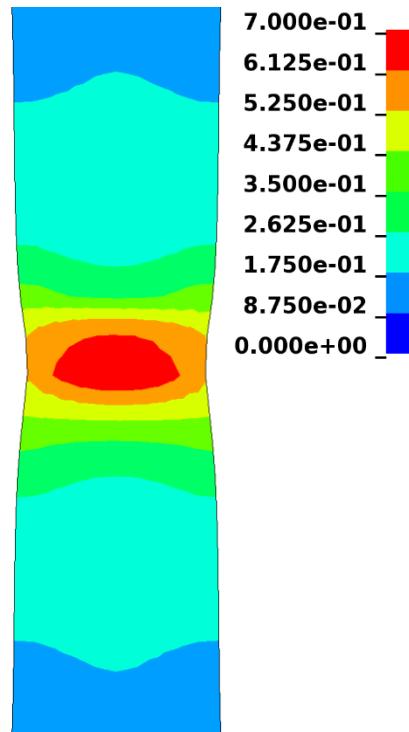
- Shell elements of formulation **ELFORM=16**
- Element size for the calibration: $L_e=0.5\text{mm}$
- Material model: ***MAT_036E, HOSF=1**
- Constant R values assumed: $R_{00}=0.7$, $R_{45}=0.5$, $R_{90}=0.8$
- Calibration through “reverse engineering”



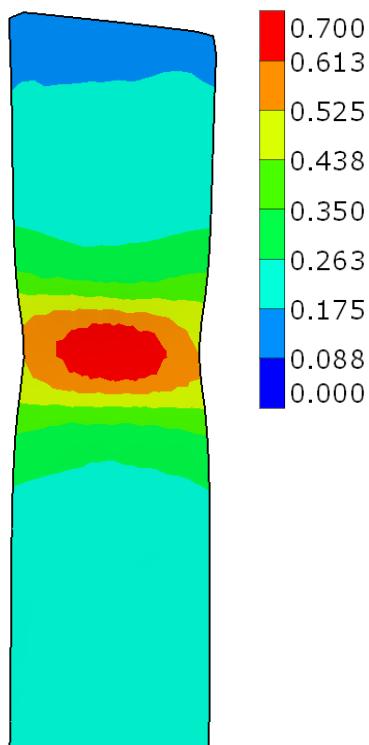
Small tensile test

Equivalent strain, rolling direction (0°)

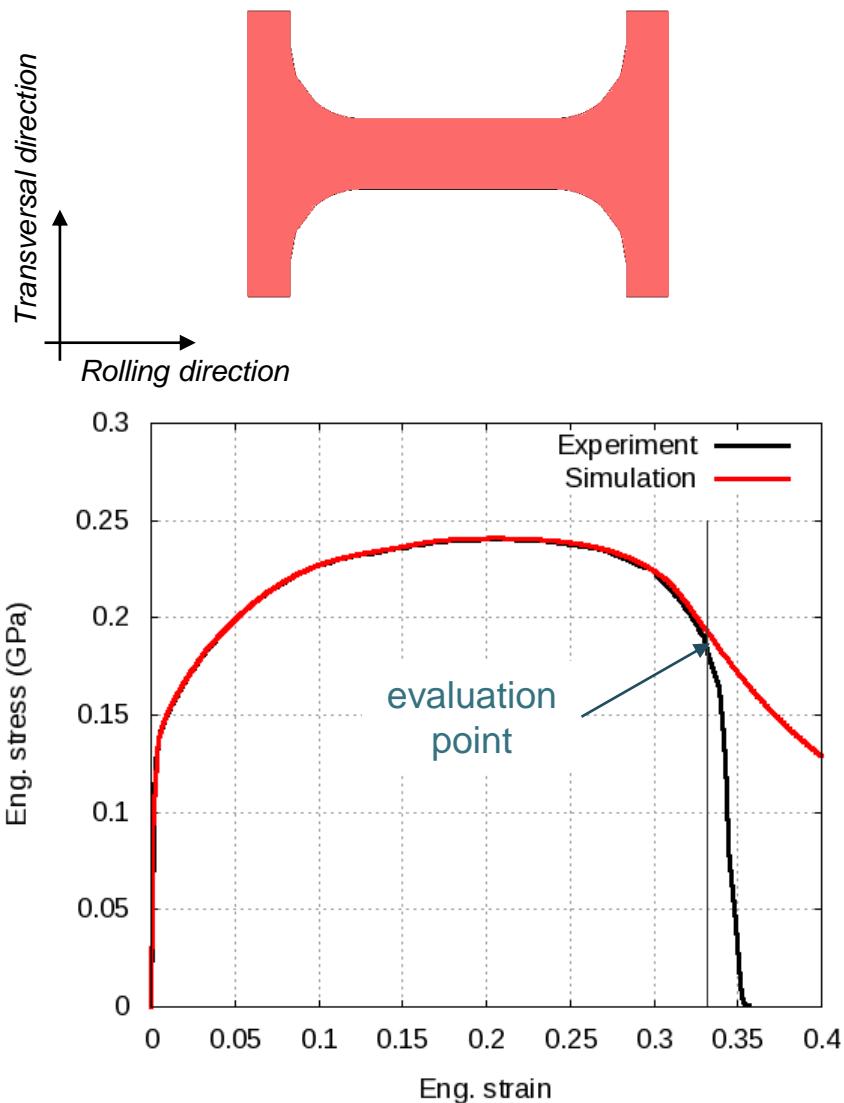
LS-DYNA
(*MAT_036E)



Experiment
(DIC with ARAMIS)



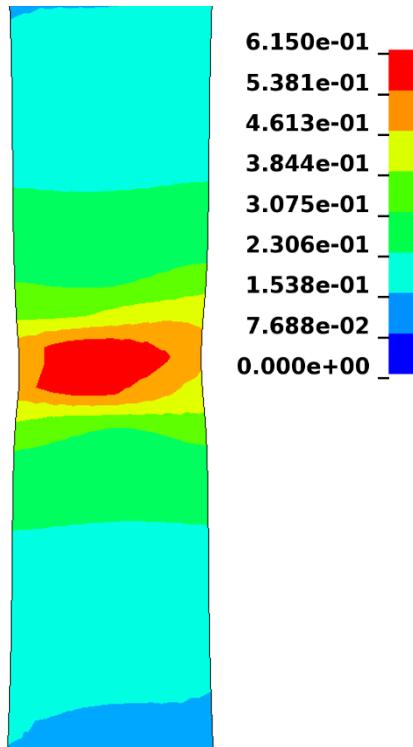
Shell elements, EFORM=16, Le=0.5mm



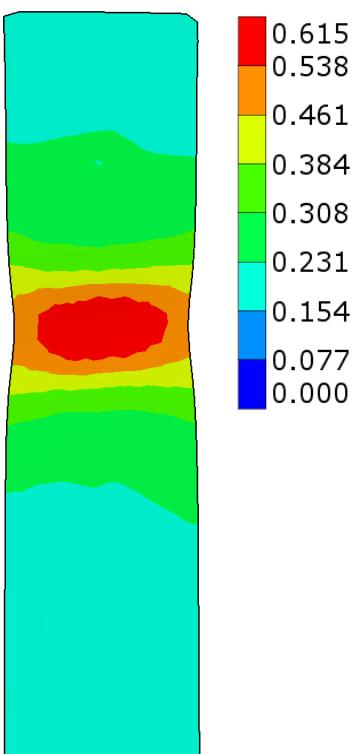
Small tensile test

Equivalent strain, diagonal direction (45°)

LS-DYNA
(*MAT_036E)



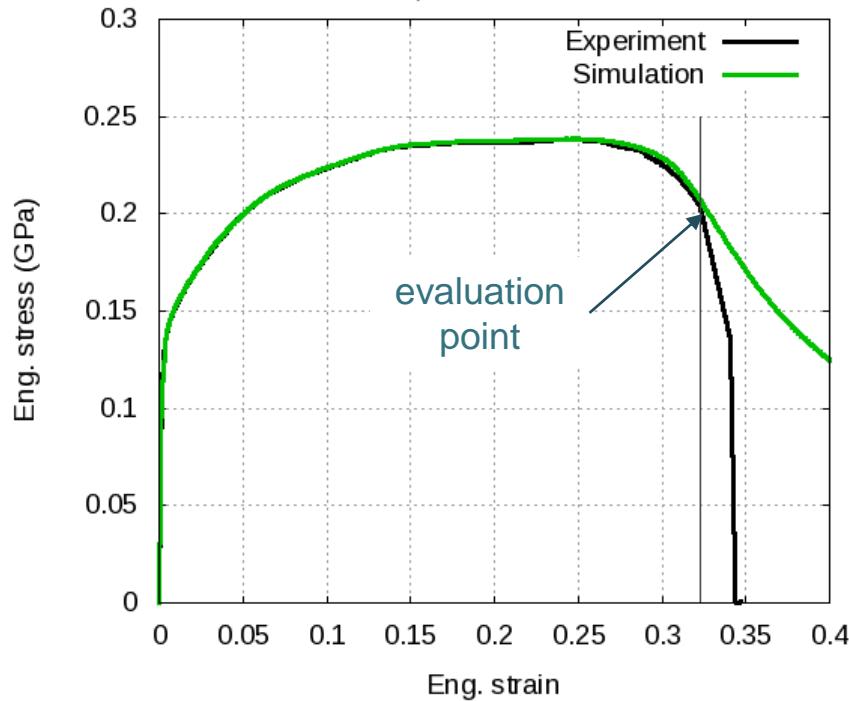
Experiment
(DIC with ARAMIS)



Shell elements, EFORM=16, Le=0.5mm

Transversal direction

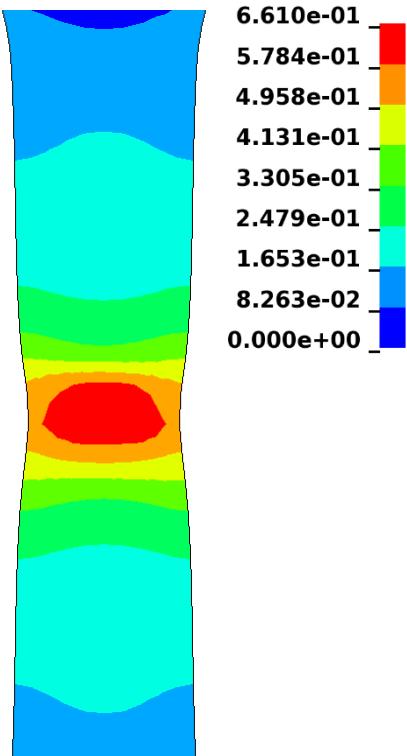
Rolling direction



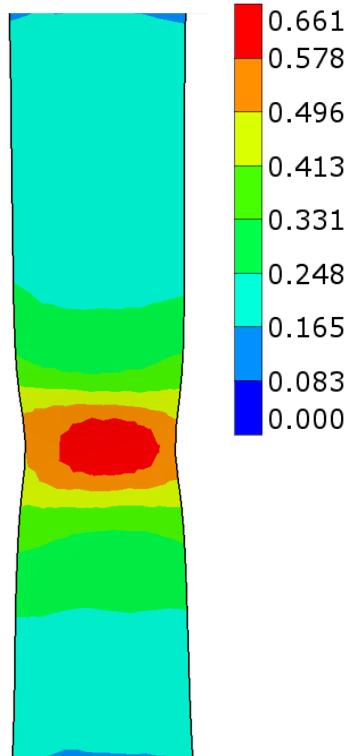
Small tensile test

Equivalent strain, transversal direction (90°)

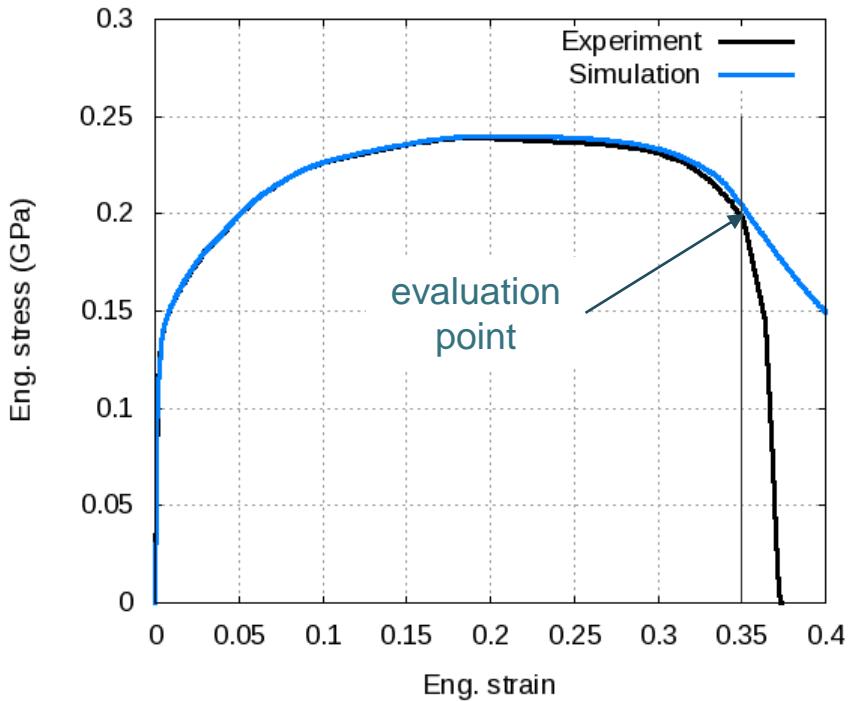
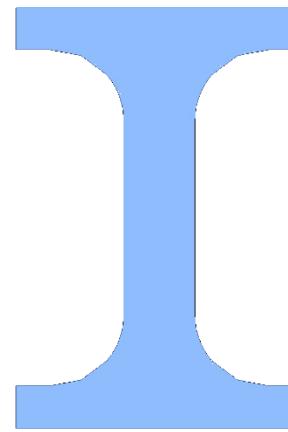
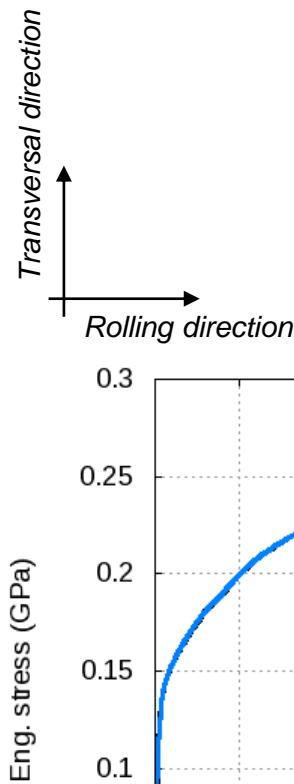
LS-DYNA
(*MAT_036E)



Experiment
(DIC with ARAMIS)

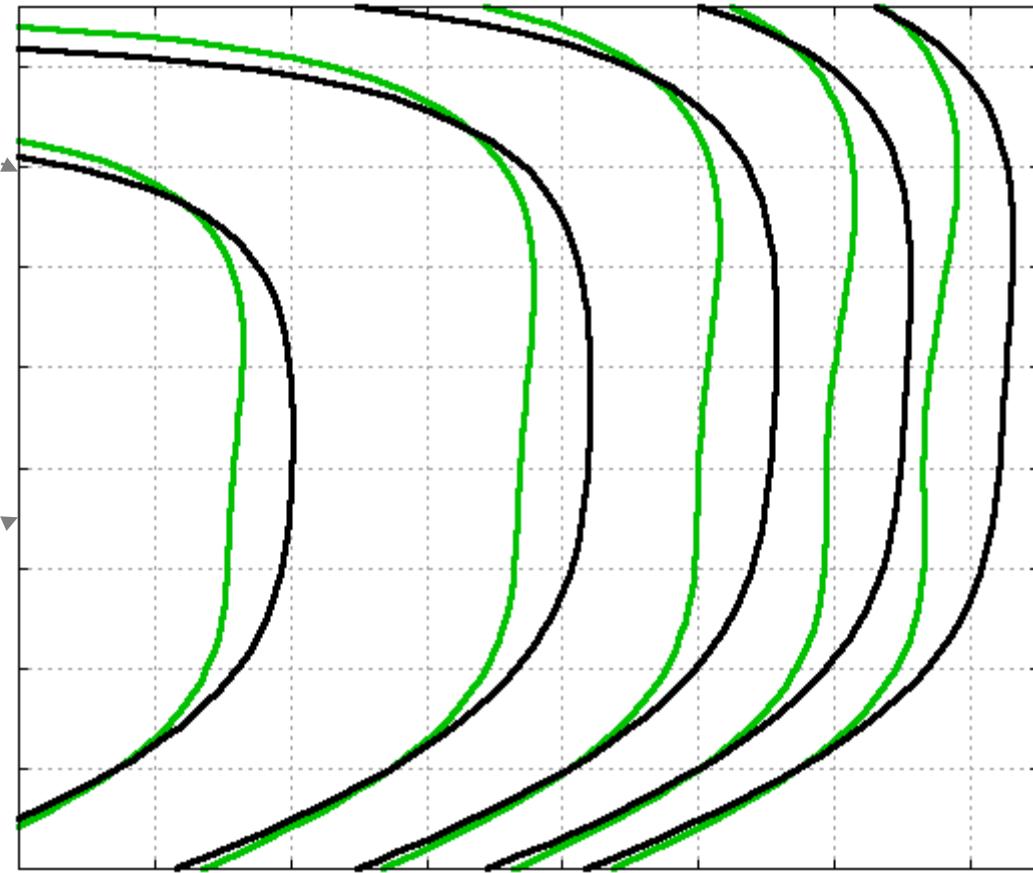
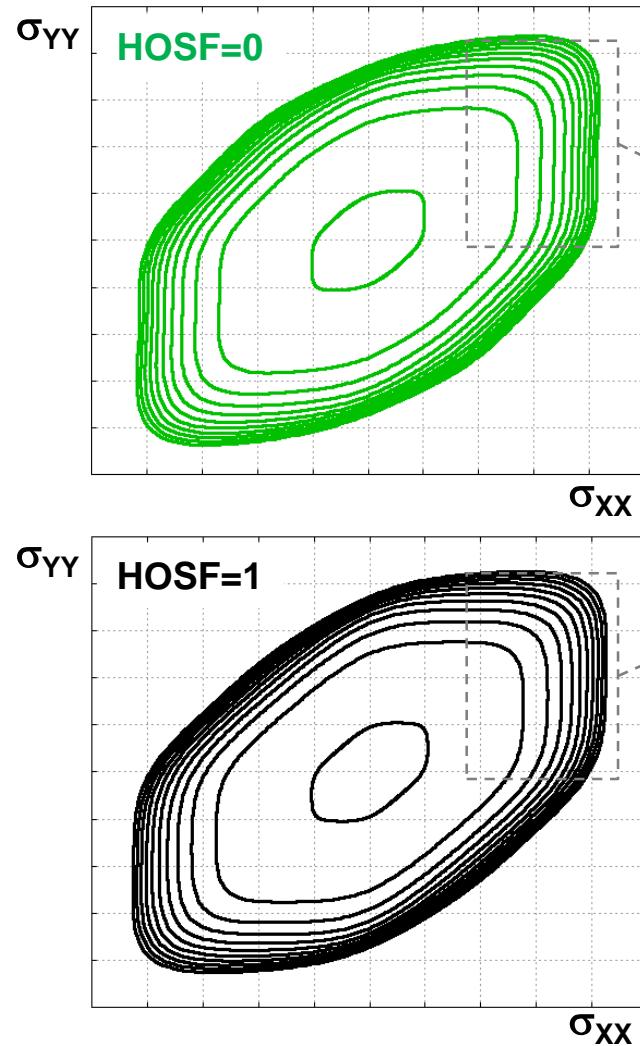


Shell elements, EFORM=16, Le=0.5mm



Yield Surface (*MAT_036E)

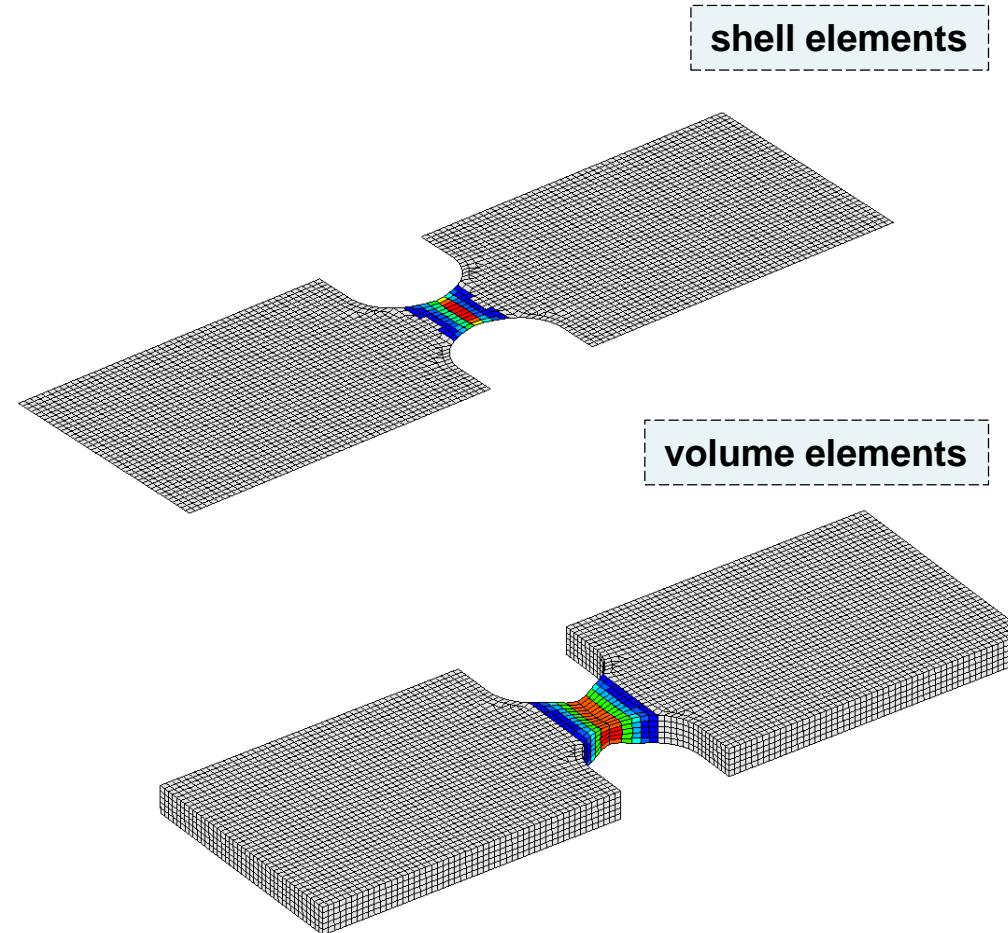
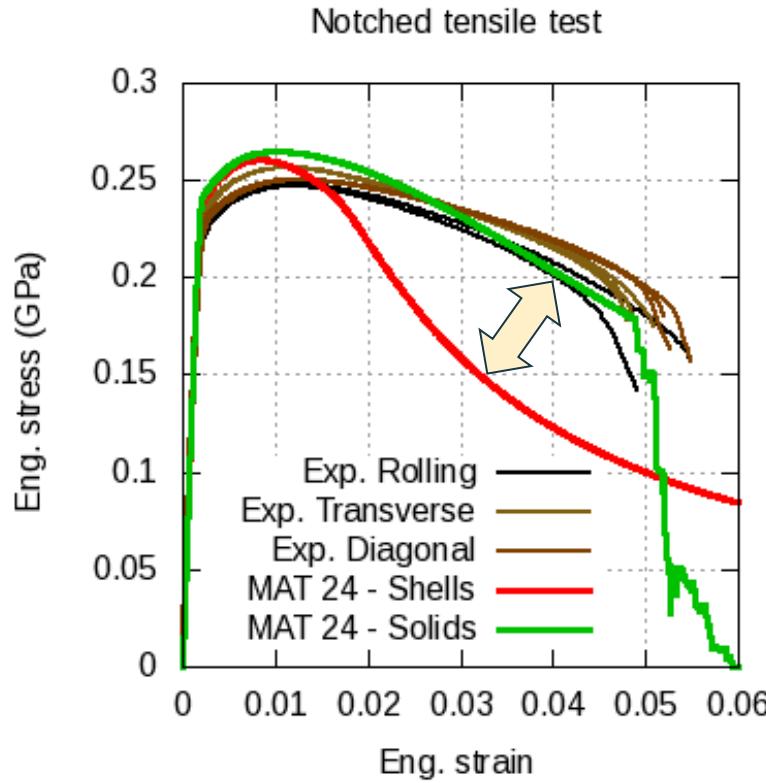
Comparison between Barlat-based (HOSF=0) and Hosford-based (HOSF=1)



Modeling with standard shell elements

Limits of the modeling approach

Example: Aluminum extrusion with $t=2.5\text{mm}$



Conclusions

Final remarks and outlook

- Good plasticity is very important for accurate local strains
(DIC measurements make validation possible)
- Plastic strain “not only coming from hardening curve”
- *MAT_036E is very flexible for matching experimental data
(concave yield surfaces still possible, but generally only for “extreme” data)
- Modeling with standard shell elements still poses some barriers...
- Extension to volume elements not quite straightforward...
... but we’re working on that too



Thank you!



Your LS-DYNA distributor and more