

D.J. Benson: Professor of Applied Mechanics, University of California, San Diego, USA

Outline

- Isogeometric Analysis
 - motivation / definition
- NURBS-based finite elements
 - NURBS / *ELEMENT_SHELL_NURBS_PATCH / elements connectivity
- Shell formulations
 continuity / rotation free shells / blended shells
- Contact with NURBS
 smoothness / example
- Current status
 MPP / examples / limits / LSPP / ...
- Todo-List
 - mass-scaling / merging / trimmed NURBS / NURBS-solids / LSPP / ...

Isogeometric Analysis – motivation & definition

- reduce effort of geometry conversion from CAD into a suitable mesh for FEA (?)
- reduce discretization errors \rightarrow adaptivity
- increase continuity of geometry representation \rightarrow contact / rot. free shells / ...

ISOPARAMETRIC (FE-Analysis)

use same approximation for geometry and deformation

GEOMETRY \leftrightarrow

DEFORMATION

ISOGEOMETRIC (CAD - FEA)

same description of the geometry in the design (CAD) and the analysis (FEA)

CAD \leftrightarrow FEA

- common geometry descriptions in CAD
 - NURBS (Non-Uniform Rational B-splines) → most commonly used
 - T-splines
 - subdivision surfaces
 - and others

- → enhancement of NURBS
- \rightarrow mainly used in animation industry

NURBS = Non Uniform Rational Basis Spline

B-spline basis functions

- recursive
- positive everywhere
- dependent on knot-vector and polynomial order
- normally C^(P-1)-continuity
- "partition of unity" (like Lagrange polynomials)
- refinement (h/p and k) without changing the initial geometry → adaptivity
- control points are normally not a part of the physical geometry (non-interpolatory basis functions)

NURBS

- B-spline basis functions + control net with weights
- all mentioned properties for B-splines apply for NURBS

• A typical NURBS-Patch and the elements

- elements are defined through the knot-vectors
- shape functions for each control-point (\rightarrow DOFs)





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- A typical NURBS-Patch and the definition of elements
 - elements are defined through the knot-vectors (interval between different values)
 - shape functions for each control-point





Shell formulations

- continuity at element boundary
 - Lagrange: C⁰
 - NURBS: C^{p-1} support of shape functions (control points) do overlap

■ FORM=0:

- shear deformable shell theory with rotational DOFs (6DOFs/control point)
- FORM=1: (requires C¹)
 rotation free shell theory without rotational DOFs (3DOFs/control point)

• FORM=4/-4:

 hybrid shell formulation combination of FORM=0 and FORM=1 (especially necessary at patch boundaries where continuity drops to C⁰)



Shell formulations – multi-patch coupling (rotation free formulation)

• G¹ continuity along patch boundaries

at patch boundaries: C⁰-lines (no transmission of bending moments with rot-free formulation)
 add rotational DOFs at patch boundary (hybrid/blended shell: FORM=4/-4)



Shell formulations – multi-patch coupling (rotation free formulation)

- keep angle at discontinuities (internal C⁰-lines)
 - no transmission of bending moments with rot-free formulation \rightarrow kinks
 - add rotational DOFS at interior control points (FORM=4)



Contact with NURBS

- all (penalty based) contacts available
 - use interpolation shell elements (lin. Quads)
 - loss of smooth (contact) surface representation
 - \rightarrow may lead to problems in tangential sliding



- NURBS contact for (SMP only)
 *CONTACT_AUTOMATIC_1WAY_S2S
 - use NURBS representation on master side
 - smooth contact behavior (i.e. tangential sliding)





Contact with NURBS – sliding example

> t

- comparison *Contact_Automatic_One_Way_Surface_To_Surface
 - IGACTC=0: master surface defined by interpolation elements (faceted)
 - IGACTC=1: master surface defined by NURBS surface (smooth)
 - _SMOOTH: smooth curve-fitted surface to represent master surface
- Example: Tube-In-Tube (both elastic bodies properties of steel)
 - frictionless sliding
 - outer tube is fixed
 - inner tube is free to move
 - inner tube with initial rotational velocity (death: t=0.01)
 - termination time: t=0.1



0.01

 $v(r_{7})$





Contact with NURBS – sliding example

computational time



	Version A		Version B		
	CPU time	#of cycles	CPU time	#of cycles	
IGACTC=0	3 min 32 sec	346535	3 min 18 sec	346484	SMP (single) ncpu=1
IGACTC=1	9 min 27 sec	346537	7 min 55 sec	346536	SMP (single) ncpu=1
_SMOOTH	8 min 43 sec	346484	6 min 10 sec (loss of contact at around t=0.03)	349561	MPP (single) -np 1

- NURBS contact (IGACTC=1)
 - insensitive to Slave-Master-Pair definition
 - more expensive than standard contact (IGACTC=0) but comparable with _SMOOTH option
 - NOTE: comparison is not representative (SMP vs. MPP), _SMOOTH option and Version B looses contact around t=0.03 (30% of computation time)

Current status - summary

Keyword: *ELEMENT_SHELL_NURBS_PATCH

- definition of NURBS-surfaces
- shell formulations with/without rotational DOFs and hybrid shell
- NURBS elements run in MPP with good scaling
- NURBS contact

(*CONTACT_AUTOMATIC_ONE_WAY_SURFACE_TO_SURFACE, IGACTC=1, SMP only)

Pre- and Postprocessing

- work in progress for LS-PrePost ... current status (Ispp4.1beta)

- → visualization of 2D-NURBS-Patches
- → import IGES-format and construct *ELEMENT_SHELL_NURBS_PATCH
- → modification of 2D-NURBS geometry
- \rightarrow ... much more to come!
- Postprocessing and boundary conditions (i.e. contact) mainly with
 - interpolation nodes (automatically created)
 - interpolation elements (automatically created)
- Analysis capabilities
 - implicit and explicit time integration
 - eigenvalue analysis
 - other capabilities (e.g. geometric stiffness for buckling) implemented but not yet tested
- LS-DYNA material library available (including umats)

Example – Vibration of a square plate



Example – dynamic buckling

- nonlinear dynamic buckling
 - isotropic elastic-plastic material
 - linear plastic hardening
 - NURBS (P=2,3,4)
 - Reissner-Mindlin formulation





Experimental results from Kyriakides

S. Kyriakides, Personal communication, 2008

- good agreement of
 - buckling load & buckling pattern
 - Reissner-Mindlin shell formulations (P=2,3,4)





Example - buckling

- standard benchmark for automobile crashworthiness
- quarter symmetry to reduce cost
- perturbation to initiate buckling mode
- J₂ plasticity with linear isotropic hardening
- mesh:
 - 640 quartic (P=4) elements
 - 1156 control points
 - 3 integration points through thickness





D.J. Benson

Example – column crush







Summary

- NURBS-based elements run stable and scale good in MPP
- higher order accurate isogeometric analysis may be cost competitive
- code optimization necessary to make it faster
- NURBS contact works well
- model creation with standard NURBS is currently the bottleneck!

ToDo-List

- perform a lot more studies in different fields \rightarrow experience
- motivate customers (and researchers) to "play" with these elements
- further implementation
 - post-processing directly with NURBS (partially available already)
 - mass scaling
 - NURBS contact implementation for MPP
 - merging of non-matching NURBS patches
 - trimmed NURBS (research currently done at UCSD)
 - make pre- and post-processing more user-friendly and faster
 - introduce 3D NURBS elements
 - ... much more

